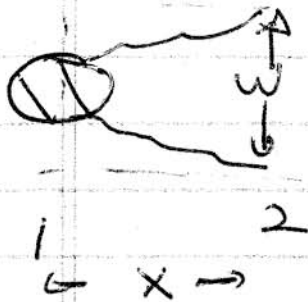


Physics 281

Lecture VI → More Wakes, Drag, Boundary Layers

→ Recall wake:



$$F_d \approx \Delta (PA)$$

$$A \sim W(x)^2$$



$$\frac{\Delta w}{R} \sim \left(\frac{x}{R}\right)^{1/2} Re^{-1/2}$$

$Re < 1$

$$\frac{w}{R} \sim \left(\frac{x}{R}\right)^{1/3} \quad Re >> 1$$

Some questions:

→ why pressure negligible? ↔ see notes

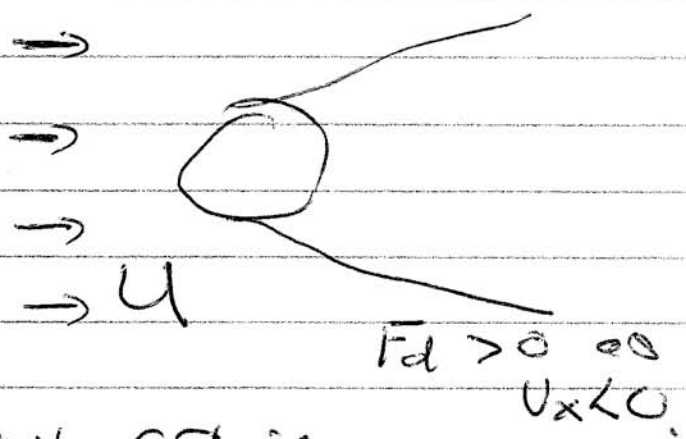
→ when does wake transition from turbulent to laminar?

Further issues/observations:

- note:

$$F_d = - \int da \rho U v_x$$

$$\approx - \rho U \int da v_x$$



deficit of fluid <sup>compensating</sup> flux with <sup>without</sup> body and

flux deficit.

$$Q = + \int da v_x$$

$$= \int da \left[ \begin{matrix} (U + v_x) \\ \text{with} \end{matrix} - \begin{matrix} U \\ \text{w/o} \end{matrix} \right]$$

Since:  $F_d = - \rho U Q$   
 $\downarrow$   
 const.

$Q$  is constant  $\Rightarrow$  indep. of  $x$ .

$\therefore$  potential flow outside wake must compensate for deficit

$\Rightarrow$  so  $\int da \cdot v$  also  $v$ -indep. outside

$\Rightarrow v \sim 1/r^2$

$\Rightarrow \phi \sim Q/r$

$\underline{v} = \nabla \phi$

↓  
stream function  
pot. flow

$\Rightarrow$  wake breaks far / aft symmetry.

Why Wake?  $\Rightarrow$  Drag

Why Drag  $\Rightarrow$  Separation!

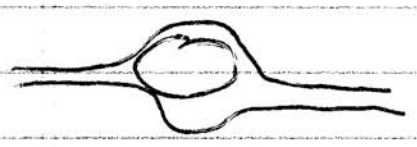
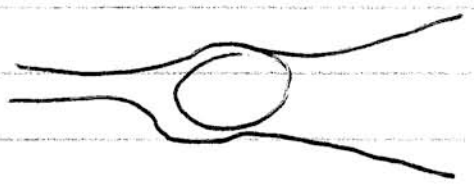
Separation

- separation occurs in boundary layer flow, which 'separates' or departs body

c.e.

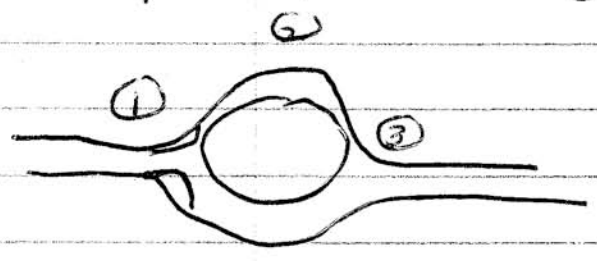
pot. flow

b.l. / separation



Separation  $\Rightarrow$  wake

Why separation?



Potential Flow:

$$P + \frac{\rho v^2}{2} = \text{const}$$

①  $P_{tot} = P_{max}$   
 $= P + \frac{\rho v^2}{2}$   
 ■  $v_x = 0$

ie. pressure maximal at stagnation point

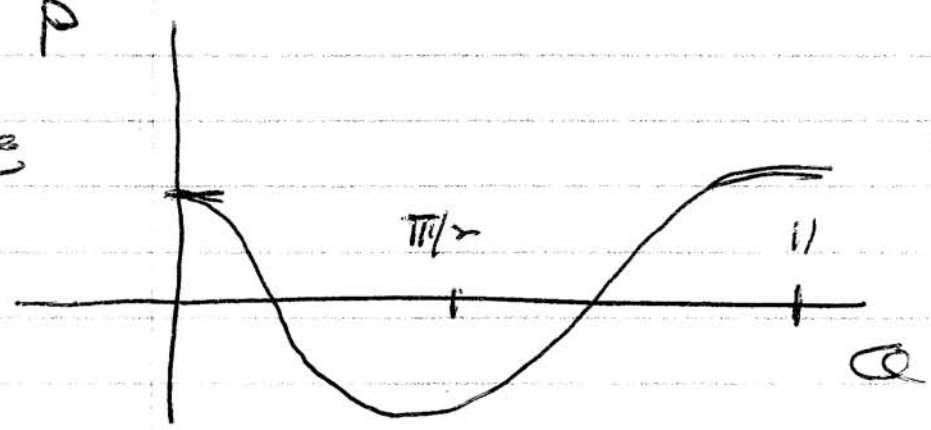
②  $v_x = v_{x, max}$   
 $P_{tot} = P_{min}$

ie. velocity relative body maximal  
ie. pressure converted to flow

③  $P_{tot} = P_{max}$   
 $= P + \frac{\rho v^2}{2}$

ie. pressure returns to maximal

if  $P$   
of flow

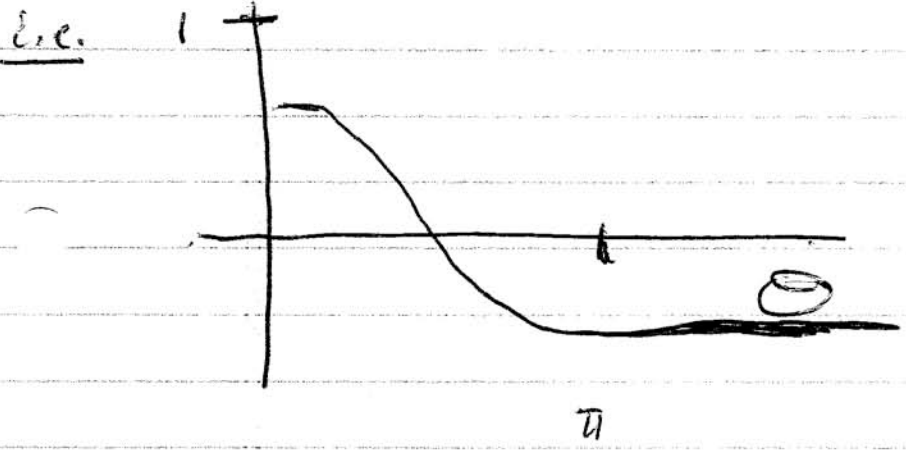


note symmetry

Now, if non-ideal fluid, no-slip

d.e. + viscosity  $\Rightarrow$  Fluid does not make it to  $\theta = \pi$ ,

d.e. Flow will stagnate prior



and



$\Rightarrow$  h.i. "separated" from body

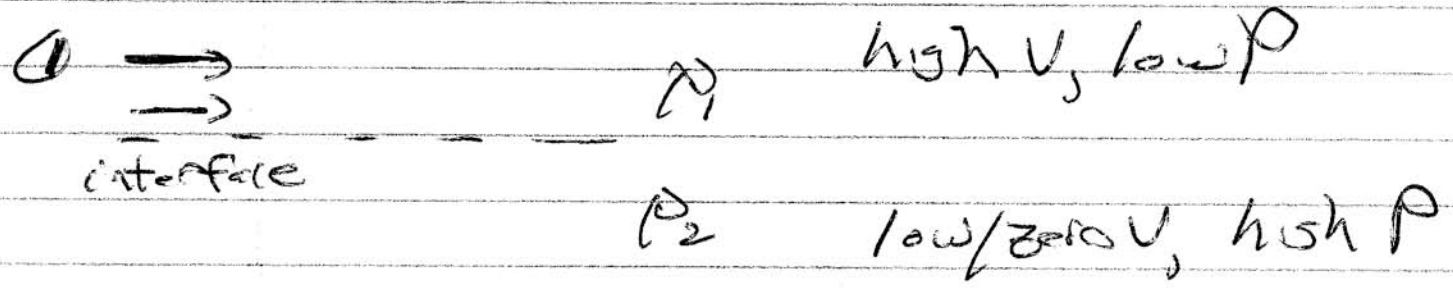
$\Rightarrow$  opens into wake

n.k. - separated flow is unstable

$\Rightarrow$  driven by  $\Delta V$  (Kelvin-Helmholtz).

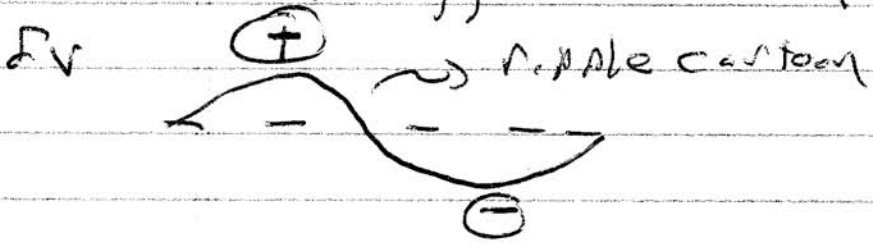
⇒ instability driven viscous, turbulent mixing

— Simple view of instability:

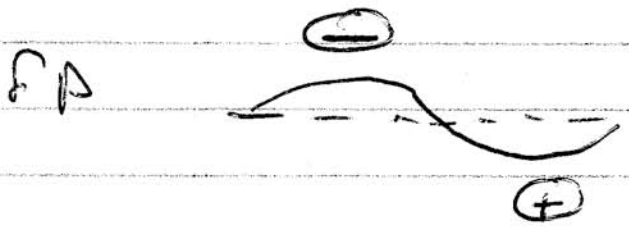


Ripple interface

② Now, consider ripple, so tweak velocity, as shown



and thus

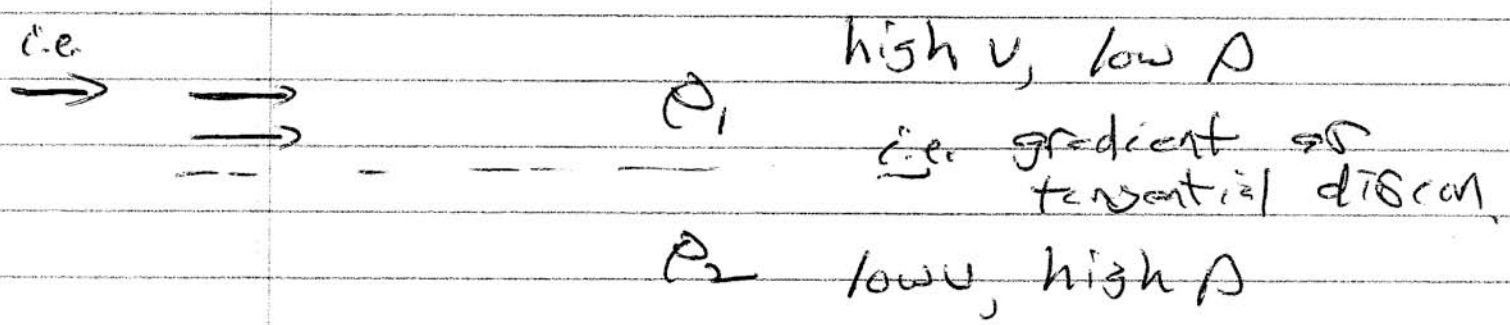


③ but  $d\rho < 0 \Rightarrow$  further  $dV > 0$

unstable

⇒ instability drives <sup>viscous</sup> turbulent mixing, etc.

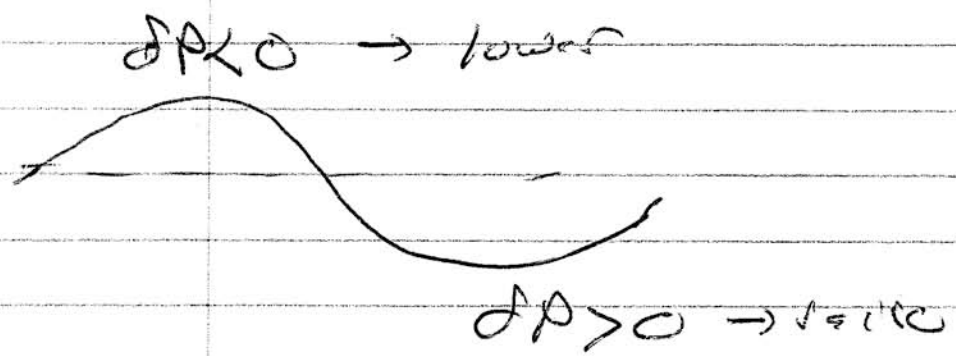
- Simple view of instability:



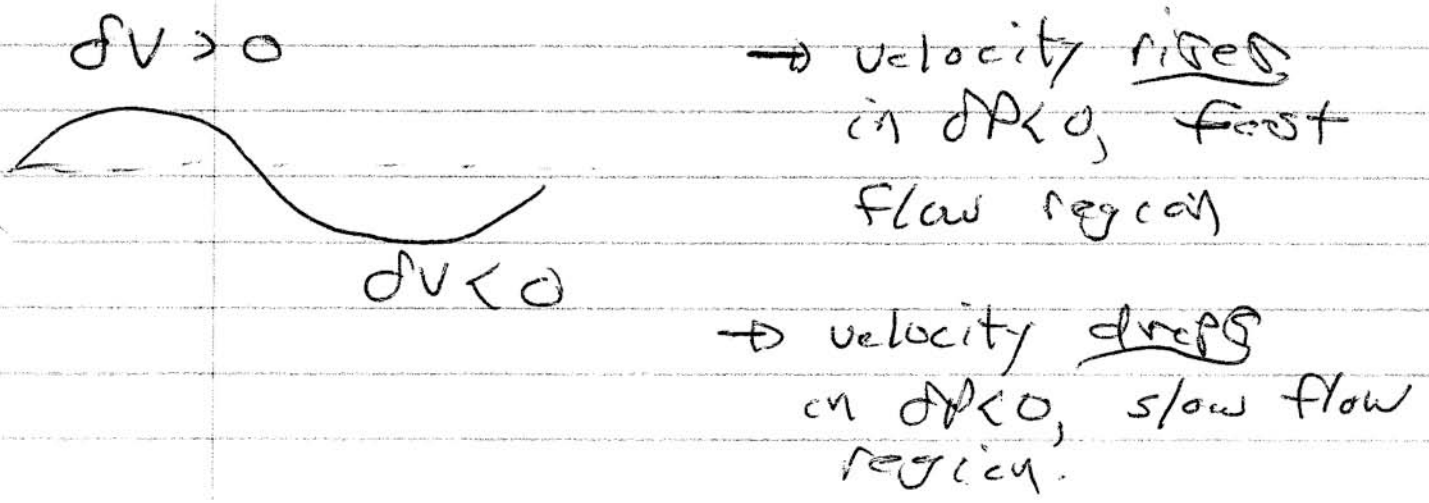
Now:

$\underline{\sigma} \cdot \underline{v} = 0$

(1) Pressure Perturbations



(2) Flow







$$\tilde{\rho}_2 = \rho_0 \exp \left[ i(kx - \omega t) - kz \right]$$

will need  $\tilde{\rho}_1(0) = \tilde{\rho}_2(0)$

then:  $\frac{\partial}{\partial t} \rho + \underline{v} \cdot \nabla \rho = -\underline{\nabla} \rho$

so

$$(-i\Omega + kv) \tilde{v}_{1z} = \frac{k \tilde{\rho}_1}{\rho_1}$$

$$\tilde{v}_{1z} = \frac{-ik \tilde{\rho}_1}{\rho(kv - \Omega)}$$

Define interface by displacement:  
 i.e. point is displ must be same

$$\frac{d\xi}{dt} = \frac{\partial \xi}{\partial t} + v \frac{\partial \xi}{\partial x} = v_z$$

so

$$-i(\Omega - kv)\xi = v_{1z}$$

comp

$$-i(\Omega - kv)\xi = \frac{-ik \tilde{\rho}_1}{\rho(kv - \Omega)}$$

∞

$$P_1 = -\frac{\rho_1}{k} (kv - \Omega)^2 \Sigma$$

and

$$P_2 = \frac{\rho_2}{k} \Omega^2 \Sigma$$

equating  $\Rightarrow$

$$\tilde{P}_1 = \tilde{P}_2$$

$$\Rightarrow -\frac{\rho_1}{\rho_2} (kV - \Omega)^2 = \frac{\rho_2}{\rho_1} \Omega^2$$

$$\Omega = kV \cdot \left( \frac{\rho_1 + i \sqrt{\rho_1 \rho_2}}{\rho_1 + \rho_2} \right)$$

$$\gamma \sim kV \frac{\sqrt{\rho_1 \rho_2}}{(\rho_1 + \rho_2)}$$

Special case

$$\gamma \approx kV/2, \quad \rho_1 = \rho_2$$

i.e.  
 $\gamma \sim kAV$   
 ↓  
 jump in velocity

→ billow formation, etc.  
 (see Falkovich, for pics)  
 upshot → mix flow, relax to V.

→ back to wake:

- instability ⇒ vortex, wake formation

- spreading, expansion.

- viscous, turbulent mixing.

## → Separation and Boundary Layers

- separation is bad  $\Rightarrow$  origin of wake drag
- separation is consequence of insufficient energy in B.L.  $\nearrow$  Flow re. Flow cut 'make it around' to rear stagnation.
- goal is to delay separation  $\Rightarrow$  maintain boundary layer attachment by
  - streamlining
  - suction
  - turbulizing the B.L.

How work?

- streamlining: tapering to reduce along stream pressure drop

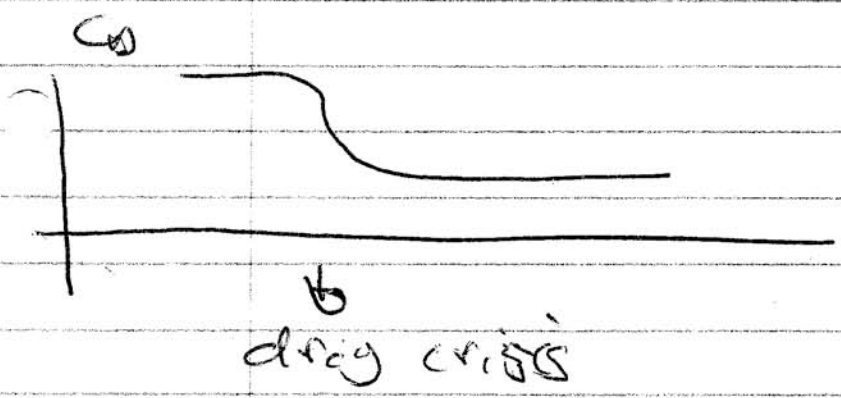


- suction: maintain attachment by sucking B.L. on, vs. detachment

- turbulizing (dimple on golf ball)

→ point is that turbulent B.L. remains attached longer (further along) by mixing energetic potential flow into B.L.

- drag with turbulent BL is lower than just before

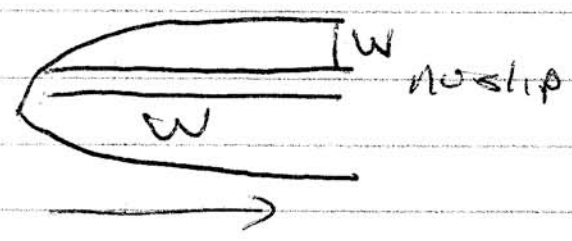
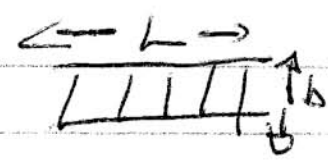


∴ this brings us to:

→ Boundary Layers

- need consider { viscous  
turbulent } BL

- Viscous (Blasius)



$$Re \sim \frac{UL}{\nu} > 1$$

$$Re \sim \frac{vW}{\nu}$$

Plate: - Blasius BL

$$- (\nu x)^{1/2} \sim w$$

$$y = \frac{x}{4}$$

$$\left(\frac{\nu x}{U}\right)^{1/2} \sim w$$

thickness

Now, for drag on plate:

$$F_d \sim \int da \cdot \rho \nu \frac{\partial v_x}{\partial y} \sim b \int_0^L dx \left( \rho \nu \frac{U}{w} \right)$$

skin friction

$$\sim b \int_0^L dx \left( \rho \nu \frac{U}{\left(\frac{\nu x}{U}\right)^{1/2}} \right)$$

$$F_d \sim -\rho v^{1/2} U^{3/2} b \int_0^L \frac{dx}{\sqrt{x}}$$

$$\sim -\rho b v^{1/2} L^{1/2} U^{3/2}$$

$$\boxed{F_d \sim -\rho v^{1/2} b L^{1/2} U^{3/2}}$$

Blasius  
Krag Law

Complete:

- Stokes  $F_d \sim -\rho \nu L U$

$Re < 1$

- Blasius  $F_d \sim -\rho v^{1/2} b L^{1/2} U^{3/2}$

$Re < 1$

$Re > 1$

- Turbulent  $F_d \sim -\rho C_D A U^2$

$Re > 1$

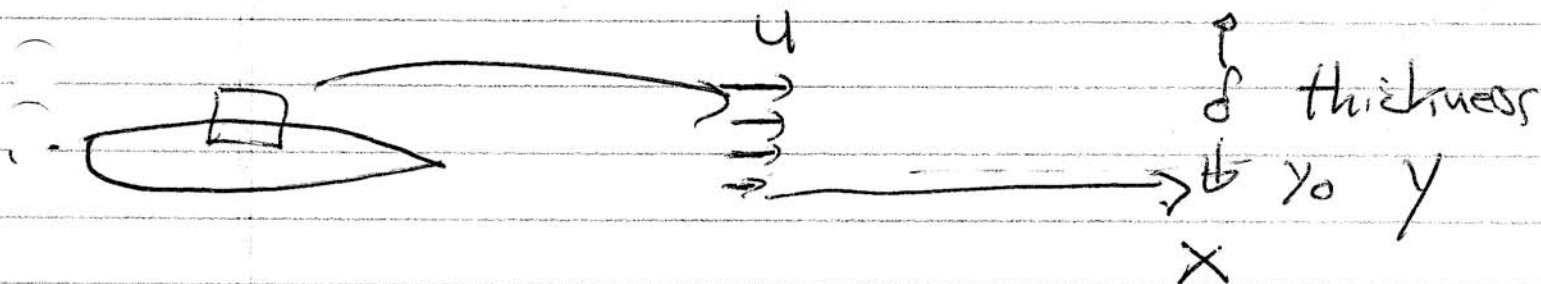
$C_D$  at large  $Re \rightarrow$  shape, etc.

→ Can extract Blasius Flow structure from approx asymptotic, aka viscous wake:

better

→ Turbulent BL

Consider  $Re \gg L$ , and  $Re \gg Re_{crit}$



Now,

→  $y_0$  is viscous sublayer → region of transition to  $V_x = 0$

→ dissipated power / volume of BL

$$P_d / V \sim \rho \frac{V^3}{y}$$

$y_0$  mixing length  
(absence scale)

(dissipn rate → indep  $M$ )



→ then, energy dispa. / time / area

$\delta \sim$  outer

$$\dot{E} \sim \int_{y_0}^{\delta} j \, dy \sim \int_{y_0}^{\delta} \frac{\rho v^3}{y} \, dy$$

$\hookrightarrow$  inner

$$\sim \rho v^3 \ln(\delta/y_0)$$

→  $y_0 \rightarrow$  laminar region  $\rightarrow$  viscous sublayer

$$y_0 \sim \nu / v \sim \nu / u$$

distinction at  $\ln \ln$  level

→ now  $\dot{E} \sim \rho v^3 \ln(\delta/y_0)$

$\downarrow$   
energy/area

$$y_0 \sim \nu / u$$

but

$$\frac{F_d}{\text{area}} \sim \rho v^2$$

Power  
 $\downarrow$

and so  $\dot{E}/\text{area} \sim \rho v^2 u \sim F_d u / \text{area}$

⇒

$$v \sim u / \ln(\delta / y_0)$$

now  $y_0 \sim \nu / u$

$$\delta \sim R \quad (\text{macro-scale})$$

$$v \sim u / \ln\left(\frac{Ru}{\nu}\right) \sim u / \ln Re$$

$$v \sim u / \ln Re$$

→  $v < u$

→ Drag is flow momentum lost to ~~body~~ <sup>body</sup>

,  $\nu$  ⇒ turbulence transports momentum to body

$$\pi_{y,x} \sim \rho \langle \tilde{v}_y \tilde{v}_x \rangle$$

$$\sim \rho v^2 \rightarrow \text{momentum flux}$$

so, for profile in BL:

$$\rho \langle u_y u_x \rangle \sim \rho v^2 \sim -\rho \tau \frac{du}{dy}$$

eddy viscosity
nb: Fickian ansatz diskutierbar

mixing velocity
diffusive flux (model)

$$v_T = v l$$

↓
mixing length

eddy / turbulent viscosity

$v \rightarrow V$  (turbulent velocity)

$l \rightarrow y$  distance from wall (none other)

$$\rho v^2 \sim \rho v y \frac{du}{dy}$$

$$du/dy \sim v/y$$

$$u \sim v \ln(y/y_0)$$

"Law of the wall"  
 → Model

⇒ logarithmic profile  
 ⇒ strong mixing.

→ thickness of BL:

$$\frac{dy}{dx} \sim \frac{v_y}{v_x}$$

$$\Rightarrow \frac{\delta}{R} \sim \frac{v}{u}$$

$$\delta \sim R / \ln(Re)$$

Thickness:  $\delta \sim R / \ln(Re) \ll R$

but compare to BL  $\delta_{BL}$

$$\delta_{BL} \sim \left( \frac{vR}{u} \right)^{1/2} \sim \frac{R}{\sqrt{Re}}$$

but  $\delta > \delta_{BL}$

$$R / \ln(Re) > R / \sqrt{Re} \quad \checkmark$$

turbulent BL much thicker than corresponding Laminar BL.

→ Key elements of turbulent BL:

- mixing process by momentum transport
- inertial layer and viscous sub-layer structure
- mixing length as  $\sim y$ , distance from wall

- balance:

$$\frac{\text{Power}}{\text{Area}} \sim \frac{F_d U}{\text{area}} \sim \int_0^{\delta} \frac{v^3}{y}$$

$$F_d/A \sim \rho v^2$$

↓  
turb. stress.

- profile of flow:



$$\rho v^2 \sim \rho v_f \frac{\partial u}{\partial y}$$

$$\sim \rho v y \frac{\partial u}{\partial y}$$

→ log profile

→ Pipe Flow

$Re \gg 1$

- similar

-  $V_*$  from:

$$\Delta P \pi R^2 \sim \rho V_*^2 2\pi R l$$

$$\frac{\Delta P \pi R^2}{l} \sim \rho V_*^2 2\pi R$$

$$- \Delta \sim \rho \frac{V_*^3}{\nu}$$

$$- \rho V_*^2 \sim \nu_T \frac{\partial u}{\partial y}$$

$$\nu_T \sim V_* y$$

PLATE 13.

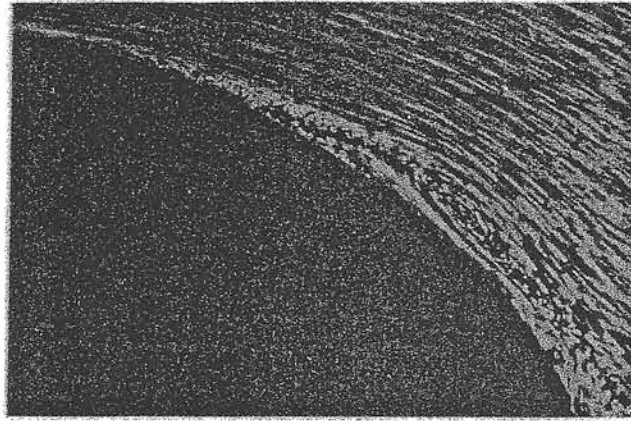


FIG. 28.

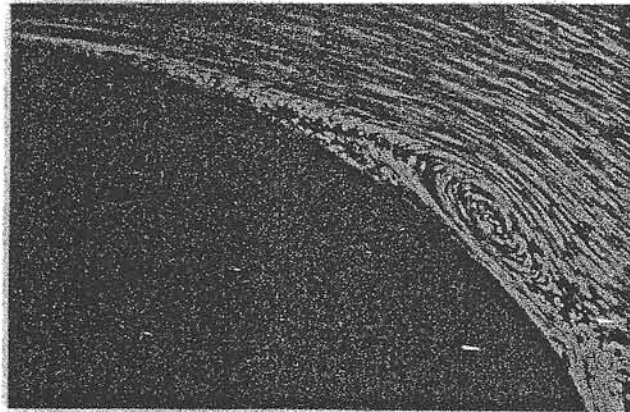


FIG. 29.

PLATE 13.—(Continued)

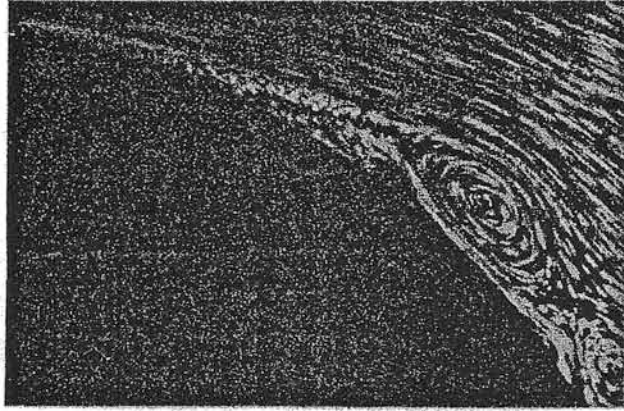


FIG. 30.

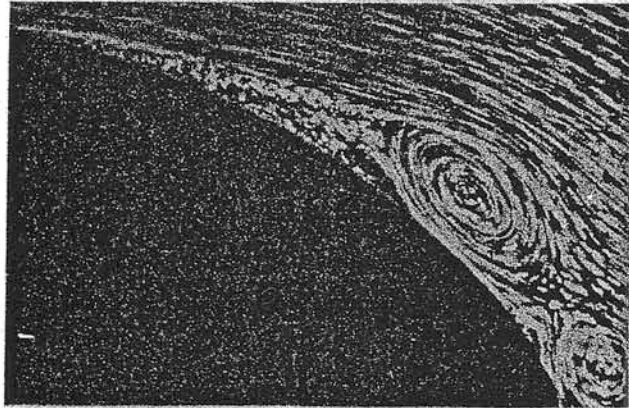


FIG. 31.



PLATE 14.

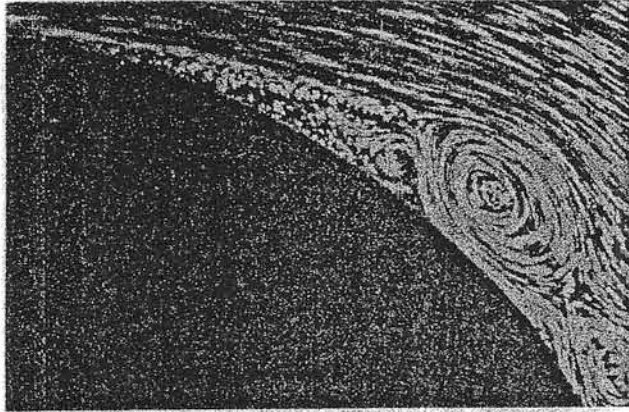


FIG. 32.

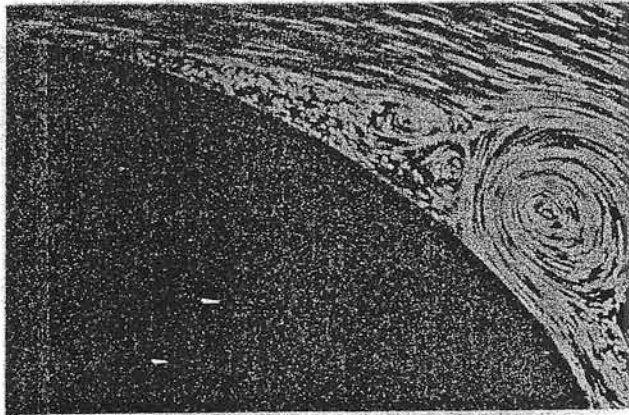


FIG. 33.

PLATE 14.—(Continued)

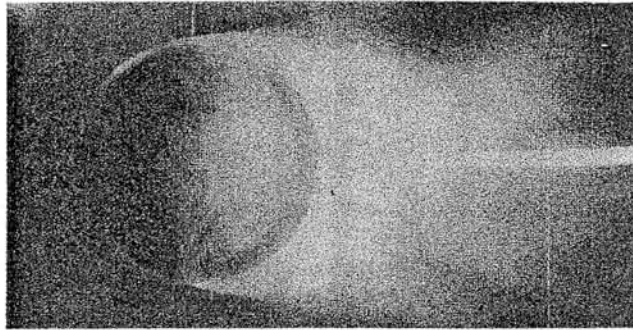


FIG. 34.—Flow round sphere below critical point. (*Wieselsberger.*)

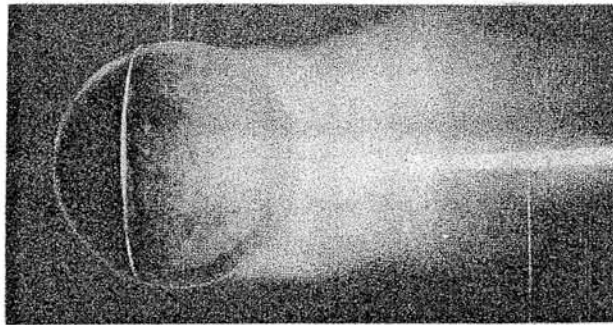


FIG. 35.—Owing to a thin wire ring round the sphere, the flow becomes of the other type with turbulent boundary layer. (*Wieselsberger.*)

PLATE 15.

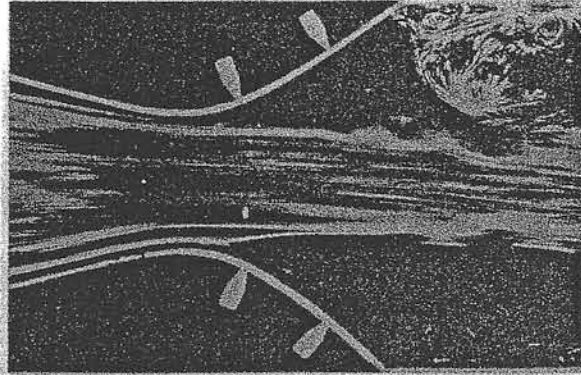


FIG. 36.—Flow in a sharply diverging channel.

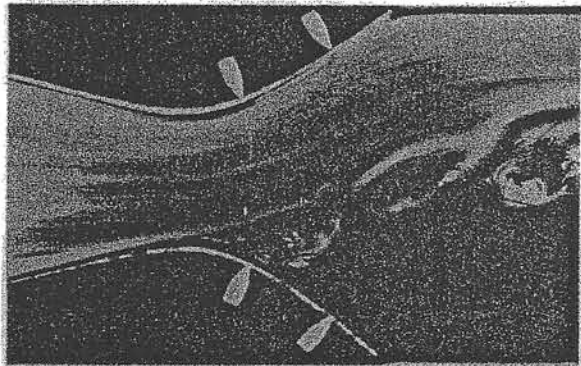


FIG. 37.—The boundary layer is sucked away at the upper wall.

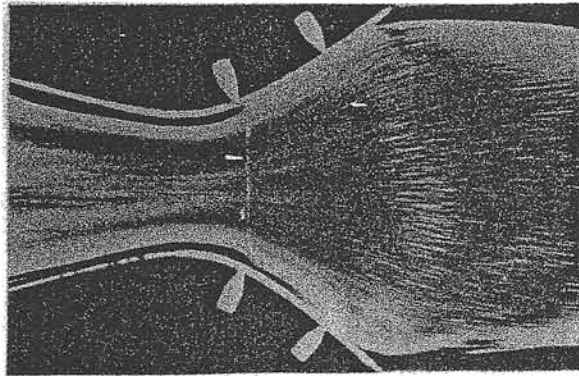


FIG. 38.—The boundary layer is sucked away on both walls; the flow is from left to right.